# ESS100 Modelling and simulation Solution to exam Tue, 14 December 2004 

## Exercise 1

(a) The implicit numerical methods have a larger stability region compared with explicit methods. Unfortunately, the implicit methods are more complex to solve.
(b) An OE-model can not be written as a linear regresion $\hat{y}=\theta^{T} \phi(t)$, because $\phi(t)$ for an OE-model will include old model output signals and these a dependent on the parameter vector $\theta$. For OE-models an iterative Gauss-Newton method is necessary.
(c) System is on standard form:

$$
\left[\begin{array}{cc}
I & 0 \\
0 & N
\end{array}\right] \dot{x}+\left[\begin{array}{cc}
-A & 0 \\
0 & I
\end{array}\right] x=\left[\begin{array}{c}
B \\
D
\end{array}\right] u
$$

The smallest integer $k$ for which $N^{k}=0$, is called the index. For our system index $=2$, because $N^{1} \neq 0$ but $N^{2}=0$.
(d) Stiff differential equations are characterized by the fact that their solutions include both fast and slow components, this usually leads to problem during simulation (time-consuming).
(e) There exist more developed methods for linear that nonlinear systems, tex frequency analysis, poles-zeros etc. The linearized model is only valid in a region closed to the stationary point around which the system has been linearized, not a global model.

## Exercise 2

For the closed loop system $y$ and $u$ can be expressed as:

$$
y=\frac{1}{1+F G}(G v+H e), \quad u=\frac{1}{1+F G}(v-F H e)
$$

or fourier transformed

$$
\begin{aligned}
Y(\omega) & =\frac{1}{1+F(i \omega) G(i \omega)}(G(i \omega) V(\omega)+H(i \omega) E(\omega)), \\
U(\omega) & =\frac{1}{1+F(i \omega) G(i \omega)}(V(\omega)-F(i \omega) H(i \omega) E(\omega))
\end{aligned}
$$

The spectrum for $u$ can than be calculated as
$\Phi_{u}(\omega)=|U(\omega)|^{2}=U(\omega) \overline{U(\omega)}=\frac{1}{|1+F(i \omega) G(i \omega)|^{2}}\left(\Phi_{v}(\omega)+|F(i \omega) H(i \omega)|^{2} \Phi_{e}(\omega)\right)$
and the cross spectrum can be calculated in a similar way
$\Phi_{y u}(\omega)=Y(\omega) \overline{U(\omega)}=\frac{1}{|1+F(i \omega) G(i \omega)|^{2}}\left(G(i \omega) \Phi_{v}(\omega)-F(i \omega)|H(i \omega)|^{2} \Phi_{e}(\omega)\right)$
The estimation of $G(p)$ can then given as

$$
\hat{G}(i \omega)=\frac{\Phi_{y u}(\omega)}{\Phi_{u}(\omega)}=\frac{G(i \omega) \Phi_{v}(\omega)-F(i \omega)|H(i \omega)|^{2} \Phi_{e}(\omega)}{\Phi_{v}(\omega)+|F(i \omega) H(i \omega)|^{2} \Phi_{e}(\omega)}
$$

If $v(t) \approx 0$ the spectrum for $v$ will be 0 , i.e. $\Phi_{v}(\omega)=0$ and the estimation of $G(p)$ then becomes

$$
\hat{G}(i \omega)=\frac{-1}{F(i \omega)}
$$

i.e. the estimation of $G(p)$ is not correct. This verifies that for system in closed loop spectral analysis does not work.

## Exercise 3

Introduce $F$ as state variable $x_{3}$. The dynamics for $x_{3}$ is according to the instruction

$$
\dot{x}_{3}=-\frac{1}{T} x_{3}+\frac{1}{T} u
$$

The dynamics for the fan process becomes:

$$
\begin{aligned}
& \dot{x}_{1}=x_{2} \\
& \dot{x}_{2}=-\frac{g}{r+L} \sin x_{1}-\frac{k}{m} x_{2}+\frac{r}{m(r+L)^{2}} x_{3} \cos x_{1} \\
& \dot{x}_{3}=-\frac{1}{T} x_{3}+\frac{1}{T} u
\end{aligned}
$$

or

$$
\dot{x}=f(x, u)
$$

Linearize around the point $\left(x_{0}=0, u_{0}=0\right)$ :

$$
\Delta \dot{x}=\left.\frac{\partial f}{\partial x}\right|_{x_{0}, u_{0}} \Delta x+\left.\frac{\partial f}{\partial u}\right|_{x_{0}, u_{0}} \Delta u=A \Delta x+B \Delta u
$$

where

$$
\left.\frac{\partial f}{\partial x}\right|_{x_{0}, u_{0}}=\left[\begin{array}{ccc}
0 & 1 & 0 \\
-\frac{g}{r+L} & -\frac{k}{m} & \frac{r}{m(r+L)^{2}} \\
0 & 0 & -\frac{1}{T}
\end{array}\right],\left.\quad \frac{\partial f}{\partial u}\right|_{x_{0}, u_{0}}=\left[\begin{array}{c}
0 \\
0 \\
\frac{1}{T}
\end{array}\right]
$$

The linear model then becomes:

$$
\Delta \dot{x}=\left[\begin{array}{ccc}
0 & 1 & 0 \\
-\frac{g}{r+L} & -\frac{k}{m} & \frac{r}{m(r+L)^{2}} \\
0 & 0 & -\frac{1}{T}
\end{array}\right] \Delta x+\left[\begin{array}{c}
0 \\
0 \\
\frac{1}{T}
\end{array}\right] \Delta u
$$

## Exercise 4

(a) Bond graph according to figure:

(b) Choose state-variables as the flow variables at the I-elements and effort variables at the C -elements. From the bond graph five state variables can be found, $v_{1}$, $v_{2}, v_{3}, F_{k 1}$ and $F_{k 2}$.

$$
\begin{aligned}
\dot{v}_{1} & =\frac{1}{m} F_{m 1}=\frac{1}{m}\left(F-\Delta F_{1}\right)=\frac{1}{m}\left(F-F_{k 1}-F_{d 1}\right)=\frac{1}{m}\left(F-F_{k 1}-d\left(v_{1}-v_{2}\right)\right) \\
\dot{v}_{2} & =\frac{1}{m} F_{m 2}=\frac{1}{m}\left(\Delta F_{1}-\Delta F_{2}\right)=\frac{1}{m}\left(F_{k 1}+d\left(v_{1}-v_{2}\right)-F_{k 2}-d\left(v_{2}-v_{3}\right)\right) \\
\dot{v}_{3} & =\frac{1}{m} F_{m 3}=\frac{1}{m} \Delta F_{2}=\frac{1}{m}\left(F_{k 2}+d\left(v_{2}-v_{3}\right)\right) \\
\dot{F}_{k 1} & =k\left(v_{1}-v_{2}\right) \\
\dot{F}_{k 2} & =k\left(v_{2}-v_{3}\right)
\end{aligned}
$$

The output signal can be derived as $L_{0}+F_{k 1} / k+F_{k 2} / k$. The state space model becomes:

$$
\begin{gathered}
\dot{x}=\left[\begin{array}{ccccc}
-d / m & d / m & 0 & -1 / m & 0 \\
d / m & -2 d / m & d / m & 1 / m & -1 / m \\
0 & d / m & -d / m & 0 & 1 / m \\
k & -k & 0 & 0 & 0 \\
0 & k & -k & 0 & 0
\end{array}\right] x+\left[\begin{array}{c}
1 / m \\
0 \\
0 \\
0 \\
0
\end{array}\right] u \\
y=L_{0}+\left[\begin{array}{lllll}
0 & 0 & 0 & 1 / k & 1 / k
\end{array}\right] x
\end{gathered}
$$

(c) Look at the structure above and augment for n wagons.

## Exercise 5

(a) Introduce a state variable at each integrator, in this case three states, $x_{1}, x_{2}$ and $x_{3}$, on the right hand side of each integrator we have the state variables and on the left hand side we have the time derivative of the state variables. From the Simulink scheme the derivatives can now be calculated as:

$$
\begin{aligned}
& \dot{x}_{1}=u-5 x_{1}+2 x_{2}+x_{3} \\
& \dot{x}_{2}=u-0.5 x_{2}+x_{3} \\
& \dot{x}_{3}=u-200 x_{3}
\end{aligned}
$$

the output signal is given as the sum of the state variables, i.e.

$$
y=x_{1}+x_{2}+x_{3}=\left[\begin{array}{lll}
1 & 1 & 1
\end{array}\right] x
$$

on state space form the model can be written as

$$
\begin{gathered}
\dot{x}=\left[\begin{array}{ccc}
-5 & 2 & 1 \\
0 & -0.5 & 1 \\
0 & 0 & -200
\end{array}\right] x+\left[\begin{array}{l}
1 \\
1 \\
1
\end{array}\right] u \\
y=\left[\begin{array}{lll}
1 & 1 & 1
\end{array}\right] x
\end{gathered}
$$

(b) Step response b belongs to the Simulink-model. The system has only real poles (A-matrix on diagonal form $=i$ diagonal elements $=$ eigenvalues) this the system non-oscillating, i.e. not response a. The slowest mode in the system will determine the transient, approximative time constant $T=2 s(T=1 / 0.5)$. From the figure step response b can now be identified, because response c has a time constant of approximately 5 s .
(c) $x_{3}$ corresponds to the fastest dynamics and is approximately 50 times faster than the other dynamics. Approximate with a static relationship $200 x_{3}=u$. Replace $x_{3}$ with $u / 200$ in the model.

The simplified model becomes

$$
\begin{gathered}
\dot{x}=\left[\begin{array}{cc}
-5 & 2 \\
0 & -0.5
\end{array}\right] x+\left[\begin{array}{l}
201 / 200 \\
201 / 200
\end{array}\right] u \\
y=\left[\begin{array}{ll}
1 & 1
\end{array}\right] x+1 / 200 u
\end{gathered}
$$

